LCM

LCR LCG

НСА

LBC CAC4 UCAC2

CAC-N UCAC-N

RCS2

RCC2

## Technical data (selection example)

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## [Specifications]

· Required clamping force: 400 N

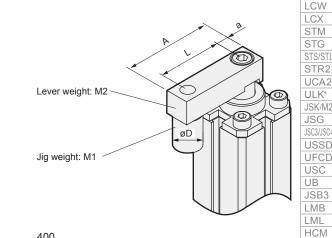
- · Working pressure: 0.5 MPa
- · Maximum piston speed: 100 mm/s

· Lever shape

M2:0.31 kg L:0.080 m A:0.1 m a:0.010 m

· Jig shape

M1:0.04 kg D:0.020 m



1. Calculate the required pressurized area.

Required pressurized area (mm<sup>2</sup>) = 
$$\frac{\text{Required clamping force (N)}}{\text{Working pressure (MPa)xefficiency}} = \frac{400}{0.5x80\%} = 1,000 \text{ (mm}^2)$$

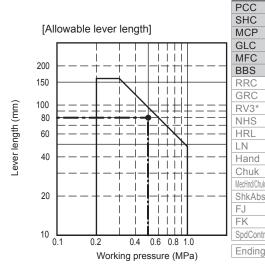
Note) Efficiency varies depending on lever length or cylinder resistance.

2. Select the cylinder size from the pressurized area (retracted side) given in the specifications list. ø40 pressurized area: 1,055 (mm²) > required pressurized area: 1,000 (mm²)

3. Confirm the allowable lever length.

Working pressure 0.5 MPa, lever length 80 mm Confirm with the graph on page 1115

→ Within usable range



4. Confirm the allowable moment of inertia for lever.

Calculating moment of inertia

(Use the formula for concentrated load on page 1114)

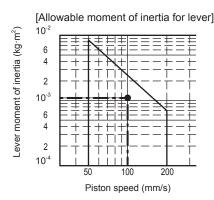
Moment of inertia I = M<sub>1</sub> (R<sub>1</sub><sup>2</sup> + K<sub>1</sub><sup>2</sup>) + 
$$\frac{M_2R_2^2}{3}$$
  
R<sub>1</sub> = L, R<sub>2</sub> = A - a, K<sub>1</sub><sup>2</sup> =  $\frac{D^2}{8}$   
I =0.04x(0.08<sup>2</sup>+ $\frac{0.02^2}{8}$ )+  $\frac{0.31x(0.1-0.01)^2}{3}$ 

=1.10x10
$$^{-3}$$
 kg·m<sup>2</sup>

Moment of inertia 1.10x10<sup>-3</sup> kg⋅m² Maximum piston speed 100 mm/s Confirm with the graph on page 1115

→ Within usable range

Based on the above, size ø40 is selected.



LCM LCR LCG LCW LCX STM STG STS/STL STR2 UCA2 ULK\* JSK/M2 JSG JSC3/JSC4 USSD UFCD USC UB JSB3 LMB LML HCM LBC CAC4 UCAC2 CAC-N UCAC-N RCS2 RCC2 PCC SHC MCP GLC MFC BBS RRC GRC RV3\* NHS HRL LN Hand Chuk MecHnd/Chuk ShkAbs FJ FΚ SpdContr Ending

## Figure for moment of inertia calculation

1_					
	Sketch	Requirements	Moment of inertia I kg·m²	Rotation radius K <sub>1</sub> <sup>2</sup>	Remarks
	Dial plate	Diameter d (m)  Weight M (kg)	$I = \frac{Md^2}{8}$	d <sup>2</sup> 8	● No mounting direction
	Bar (center of rotation at end) Gircular stepped plate	<ul> <li>Diameter d₁(m) d₂(m)</li> <li>Weight d₁ Part M₁ (kg) d₂ Part M₂ (kg)</li> </ul>	$I = \frac{1}{8} (M_1 d_1^2 + M_2 d_2^2)$	d <sub>1</sub> <sup>2</sup> +d <sub>2</sub> <sup>2</sup> 8	■ Ignore when the d₂ section is extremely small compared to the d₁ section
	Bar (center of rotation at end)	● Bar length R (m) ● Weight M (kg)	$I = \frac{MR^2}{3}$	R <sup>2</sup> 3	● The mounting direction is horizontal.
	Thin rod		$I = \frac{M_1 \cdot R_1^2}{3} + \frac{M_2 \cdot R_2^2}{3}$	$\frac{R_1^2 + R_2^2}{3}$	● The mounting direction is horizontal.
	Bar (center of rotation / gravity)	● Bar length R (m) ● Weight M (kg)	$I = \frac{MR^2}{12}$		● No mounting direction
Thin rectangle plate	(rectangular parallelepiped)	<ul> <li>Plate length</li> <li>a<sub>1</sub></li> <li>a<sub>2</sub></li> <li>Side length</li> <li>Weight</li> <li>M<sub>1</sub></li> <li>M<sub>2</sub></li> </ul>	$I = \frac{M_1}{12} (4a_1^2 + b^2) + \frac{M_2}{12} (4a_2^2 + b^2)$	(4a <sub>1</sub> <sup>2</sup> +b <sup>2</sup> )+(4a <sub>2</sub> <sup>2</sup> +b <sup>2</sup> ) 12	● The mounting direction is horizontal.
	Rectangular parallelepiped (rectangular parallelep	● Side length a (m) ● Weight b (m)  M(kg)	$I = \frac{M}{12}(a^2 + b^2)$	<u>a²+b²</u> 12	No mounting direction
	Concentrated load  R <sub>2</sub> Arm M <sub>2</sub>	centrated M <sub>1</sub> Shape of concentrated load  Length to center of gravity of concentrated load R <sub>1</sub> Arm length R <sub>2</sub> (m)  Concentrated load weight M <sub>1</sub> (kg)  Arm weight M <sub>2</sub> (kg)	$I = M_1(R_1^2 + k_1^2) + \frac{M_2 R_2^2}{3}$	Calculate k <sub>1</sub> <sup>2</sup> according to shape of concentrated load	<ul> <li>The mounting direction is horizontal.</li> <li>When M<sub>2</sub> is extremely small compared to M<sub>1</sub>, it may be calculated as M<sub>2</sub> = 0</li> </ul>