

# FCK

## Related Components Shock Absorber, Adjustable Type



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Related Equipment

SKL

NCK

SCK

FCK

FJ

FK

Cylinder Switch

Ending

Related Equipment

SKL

NCK

SCK

FCK

FJ

FK

Cylinder Switch

Ending

## Selectable from 3 stages: Low, Medium, High Speed

### 3 stages of impact speed, 3 types of mechanism

By changing the orifice structure, it is equipped with absorption functions suitable for low, medium, and high impact speeds.

### With Rotation Stopper

Lock screw adopted to prevent adjuster from becoming misaligned during use. (Not available on some compact models)

### Easy mounting with outer diameter thread

Outer diameter threads M10 to M27 for low speed, M10 to M42 for medium/high speed, with nut. Mounting and position adjustment are easy.



**Shock Absorber**  
**5 Merits**

- Safely stops the colliding object
- Increases manufacturing cycle.
- Life of machinery is extended
- Reduces noise and improves the environment around machinery
- Prevents machine breakdown

## Shock Absorber FCK Series with **32** models available

Optimal shock absorption matching impact conditions/ characteristics is possible

### 8 Low Speed Models

- Colliding speed range: 0.3 to 1 m/s
- Max absorbed energy 1.5 to 79.3 J
- Single-hole Orifice Structure



### 12 Medium Speed Models

- Colliding speed range: 0.3 to 2 m/s
- Max absorbed energy 1.8 to 720 J
- Multi-hole Irregular Orifice Structure



### 12 High Speed Models

- Colliding speed range: 0.7 to 3 m/s
- Max absorbed energy 1.8 to 720 J
- Multi-hole Orifice Structure



### Option

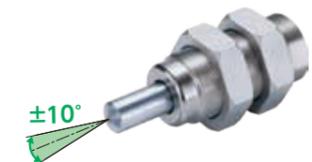
#### Stopper Nut

Positioning is possible with stopper nut.



#### Offset Angle Adapter

Absorbs offset angle of Max  $\pm 10^\circ$  at workpiece impact.



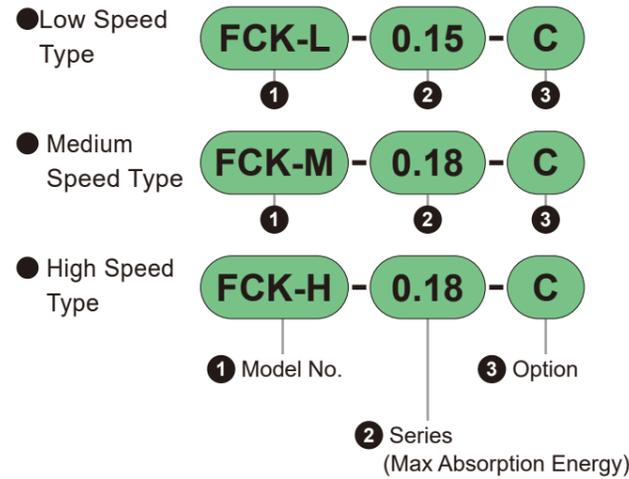


# Shock Absorber FCK Series

● Max Absorbed Energy: 1.5 to 720 J



## Model Number Notation



Rechargeable Battery Compatible Specification (Catalog No. CC-1226AA)

● Design compatible with rechargeable battery manufacturing process

**FCK - . . . . - P4\***

\* Please contact us for details.

### ① Model No.

Code	Content
<b>FCK-L</b>	Low Speed Type
<b>FCK-M</b>	Medium Speed Type
<b>FCK-H</b>	High Speed Type

### ③ Option

Code	Content
<b>Blank</b>	Without Tip Cap
<b>C</b>	With Tip Cap

### ② Series (Max Absorption Energy)

Code	Content	Low Speed Type	Medium Speed Type	High Speed Type
		FCK-L	FCK-M	FCK-H
<b>0.15</b>	1.5J	●		
<b>0.18</b>	1.8J		●	●
<b>0.3</b>	2.9J	●		
<b>0.4</b>	3.9J	●		
<b>0.5</b>	4.9J		●	●
<b>0.6</b>	5.9J		●	●
<b>1</b>	9.8J	●	●	●
<b>3</b>	29.4J	●	●	●
<b>5</b>	49J	●	●	●
*1 <b>6.5</b>	63.7J	●	●	●
*1 <b>8.1</b>	79.3J	●	●	●
<b>20</b>	196J		●	●
<b>40</b>	392J		●	●
*1 <b>45</b>	441J		●	●
*1 <b>73.5</b>	720J		●	●

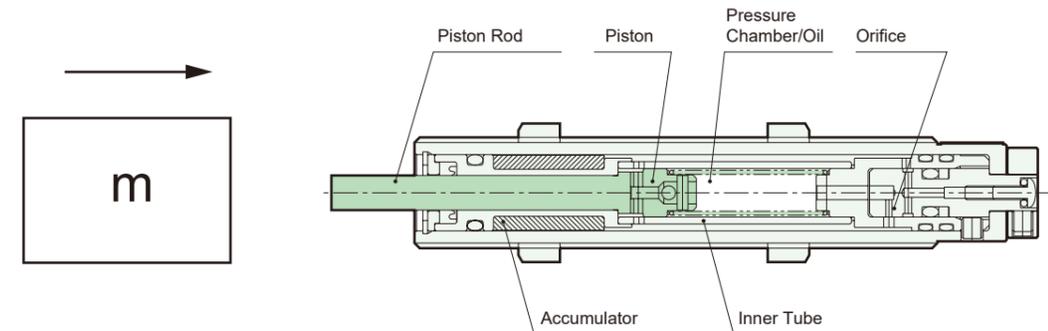
\*1: Without cap is not available for 6.5 (63.7J), 45 (441J), 73.5 (720J).

■ indicates not manufacturable.

## Specifications

Item	FCK																
Series	0.15	0.18	0.3	0.5	0.4	0.6	1	3	5	6.5	8.1	20	40	45	73.5		
Model / Classification	Adjustable spring return type																
Maximum absorption energy J	1.5	1.8	2.9	4.9	3.9	5.9	9.8	29.4	49	63.7	79.3	196	392	441	720		
Outer diameter thread size mm	M10×1.0		M12×1.0		M14×1.5		M16×1.5	M20×1.5	M25×1.5		M27×1.5	M30×1.5	M36×1.5	M42×1.5			
Stroke mm	8		10				12	16	30	40	25	35	50		80		
Max Absorption Energy per Hour kJ/hour	3.5		5.9		8.8		14.1	20.6	29.4	38.2	32.3	70.5	141.1	164.6	264.6		
Maximum collision speed	L m/s	0.3 to 1	-	0.3 to 1	-	0.3 to 1	-	0.3 to 1						-	-	-	-
	M m/s	-	0.3 to 2	-	0.3 to 2	-	0.3 to 2	0.3 to 2						0.3 to 2			
	H m/s	-	0.7 to 3	-	0.7 to 3	-	0.7 to 3	0.7 to 3						0.7 to 3			
Max Repetition Frequency (20°C) cycles/min	60											30	10	6			
Ambient Temperature °C	-5 to 70 (provided there is no freezing)																
Max Load (Resistance Force Value)	L N							2.646	4.900								
	M N	637		1,470		1.813		2.646	3.528	3.920	6.370	16.660	23.520	27.028			
	H N																
Return Time s	0.5 or less											1 or less		2 or less			
Weight	Without cap g	26.5	44	68	108	180	406	-	411	710	1300	-	-				
	With cap g	27	47	73	117	202	436	459	460	760	1410	1560	2010				
Return Spring Force	At Extension N	2.9	4.9	4.5	5.4	12.0	16.6	23.8	16.2	19.6	22.5	24.5					
	At Compression N	5.9	9.8		14.7		18.0	33.1	71.4	27.2	44.1	68.6	83.3	98.0			

## Operational Principle



When an object collides with the Piston Rod, its motion is transmitted to the oil in the pressure chamber enclosed by the piston and inner tube. The oil in the pressure chamber flows out from the orifice provided in the inner tube. At that time, a resistance force  $F$  represented by the following formula is generated.

$$F = av^2 + bv + cx$$

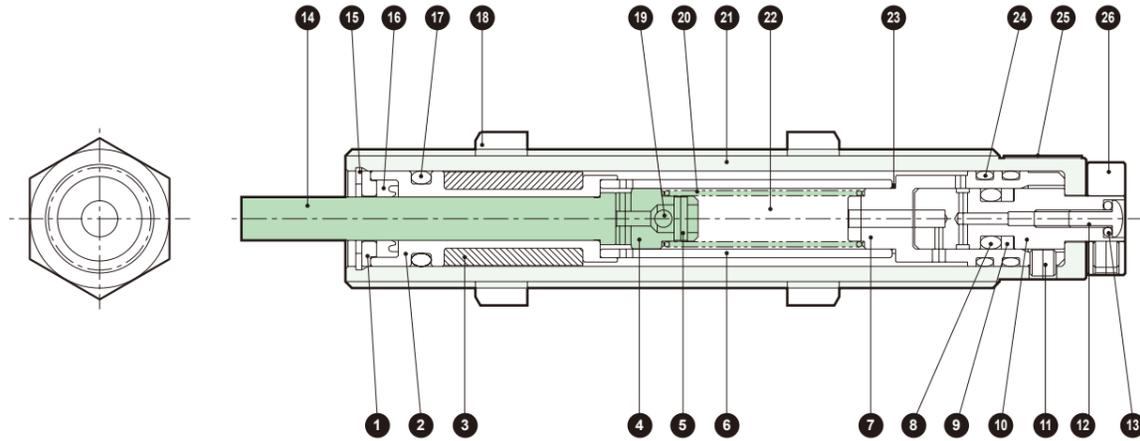
( $v$  indicates impact speed,  $x$  indicates travel stroke.  $a$ ,  $b$ ,  $c$  are constants)

The first term represents velocity squared resistance and accounts for a large weight within the resistance force. The second term represents viscous resistance and accounts for a large proportion when the impact speed is small. The third term represents the Piston Rod return force. (Since it is extremely small compared to the first and second terms, it can usually be ignored.) The product of this generated reaction force and the stroke of the Piston Rod becomes the absorption energy of the shock absorber. The shock absorber achieves ideal shock absorption by controlling the first and second terms.

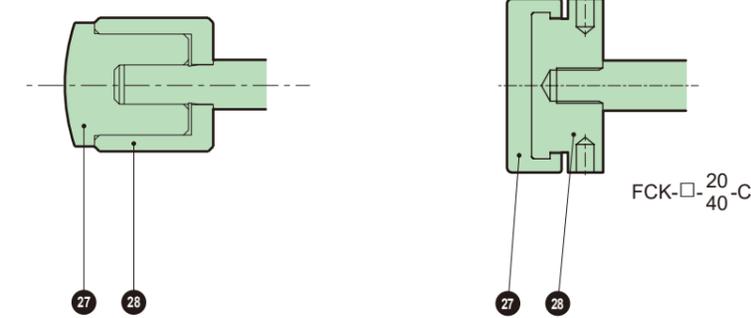


● FCK-□-□

Basic Type (Without Tip Cap)



With Tip Cap

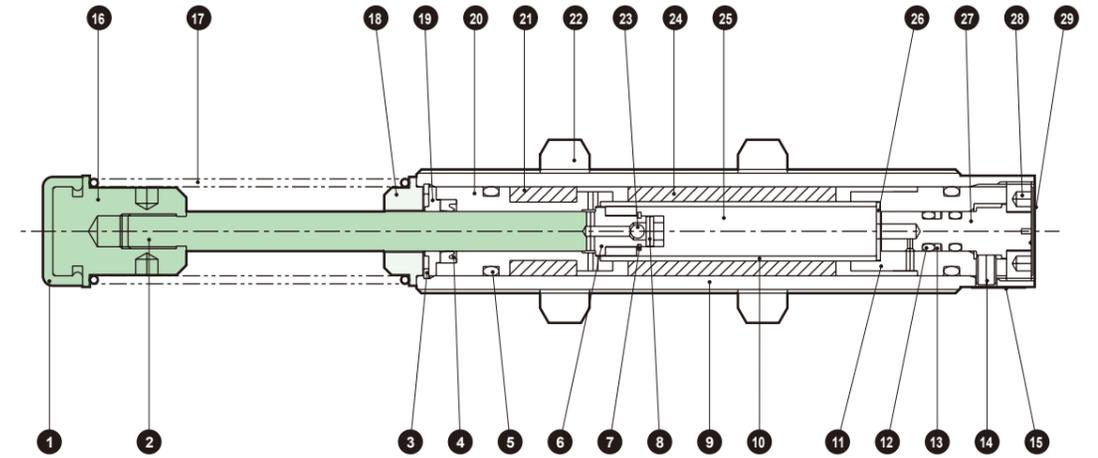


**Do not disassemble**

Part No.	Part Name	Material	Part No.	Part Name	Material
1	Packing Retainer	Copper Alloy	15	Retaining Ring	Steel
2	Guide	Copper Alloy	16	U-Packing	Nitrile Rubber
3	Accumulator	Nitrile Rubber	17	O-ring	Nitrile Rubber
4	Piston	Copper Alloy	18	Hexagon Nut	Steel
5	Spring pin	Stainless Steel	19	Steel ball	Bearing Steel
6	Inner Tube	Steel	20	Spring	Piano Wire
7	Bottom	Copper Alloy	21	Outer Tube	Steel
8	O-ring	Nitrile Rubber	22	Oil	Oil
9	Backup Ring	Resin	23	Spacer	Nitrile Rubber
10	Adjusting Shaft	Copper Alloy	24	O-ring	Nitrile Rubber
11	Hexagon Socket Set Screw	Alloy Steel	25	Product Nameplate	
12	Phillips Head Set Screw	Alloy Steel	26	Knob	Copper Alloy
13	O-ring	Nitrile Rubber	27	Rod Cap	Resin *2
14	Piston Rod	Alloy Steel	28	Reinforcing Ring	Steel

\*1: Structure varies slightly depending on the model.  
\*2: Size 20, 40 are Urethane Rubber.

6.5  
● FCK-□- 45 -C (Capped)  
73.5



**Do not disassemble**

Part No.	Part Name	Material	Part No.	Part Name	Material
1	Rod Cover	Urethane Rubber (*2)	16	Spring Guide	Steel
2	Piston Rod	Alloy Steel	17	Spring	Piano Wire
3	Retaining Ring (Round R Type)	Steel	18	Spring Guide	Steel
4	U-Packing	Nitrile Rubber	19	Packing Retainer	Copper Alloy
5	O-ring	Nitrile Rubber	20	Guide	Copper Alloy
6	Piston	Copper Alloy	21	Accumulator	Nitrile Rubber
7	Retaining Ring (E-type)	Steel	22	Hexagon Nut	Steel
8	Spring pin	Stainless Steel	23	Steel ball	Bearing Steel
9	Outer Tube	Steel	24	Accumulator	Nitrile Rubber
10	Inner Tube	Steel	25	Oil	Oil
11	Bottom	Copper Alloy	26	Washer	Steel
12	O-ring	Nitrile Rubber	27	Adjusting Shaft	Copper Alloy
13	Backup Ring	Resin	28	Retainer Screw	Steel
14	Hexagon Socket Set Screw	Alloy Steel	29	Adjustment Label	Steel
15	Product Nameplate				

\*1: Structure varies slightly depending on the model.  
\*2: Size 45, 73.5 do not come with a rod cover (resin) cap.



Shock Absorber FCK Series Optional Parts

# FCK-□-N1, FCK-□-C-N1

(Stopper Nut)



Shock Absorber FCK Series Optional Parts

# FCK-□-A

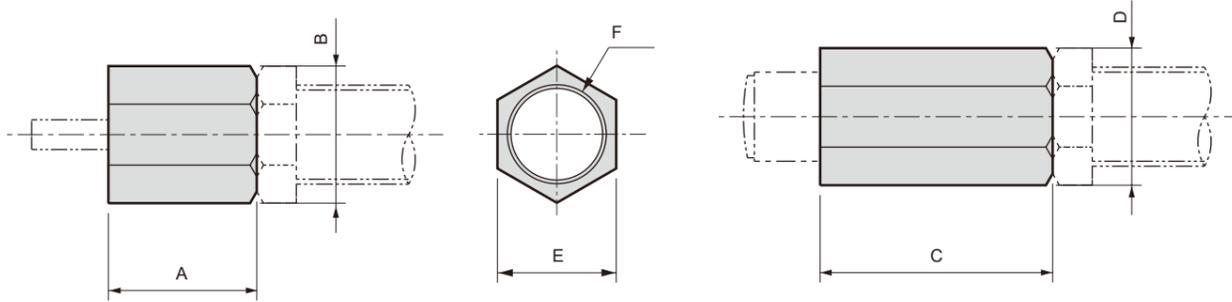
(Offset Angle Adapter)



## Dimensional Drawings

● FCK-□-N1 (standard)  
Material: Steel

● FCK-□-C-N1 (capped)  
Material: Steel



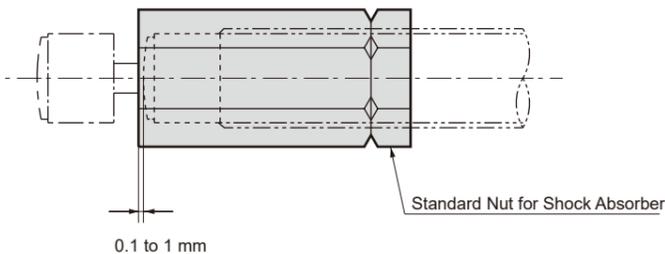
(For Standard Type)

(For Type with Cap)

	Model No.	Applicable Models	A	B	Weight (g)	Model No.	Applicable Models	C	D	E	F	Weight (g)
FCK	FCK-0.18-N1	FCK-L-0.15 FCK-M-0.18 FCK-H-0.18	10	15	5	FCK-0.18-C-N1	FCK-L-0.15-C FCK-M-0.18-C FCK-H-0.18-C	16	15	13	M10×1	8
FJ	FCK-0.5-N1	FCK-L-0.3 FCK-M-0.5 FCK-H-0.5	12	16.2	5	FCK-0.5-C-N1	FCK-L-0.3-C FCK-M-0.5-C FCK-H-0.5-C	16	16.2	14	M12×1	7
FK	FCK-0.6-N1	FCK-L-0.4 FCK-M-0.6 FCK-H-0.6	12	19.6	9	FCK-0.6-C-N1	FCK-L-0.4-C FCK-M-0.6-C FCK-H-0.6-C	20	19.6	17	M14×1.5	15
	FCK-1-N1	FCK-L-1 FCK-M-1 FCK-H-1	15	21.9	13	FCK-1-C-N1	FCK-L-1-C FCK-M-1-C FCK-H-1-C	30	21.9	19	M16×1.5	26
	FCK-3-N1	FCK-L-3 FCK-M-3 FCK-H-3	30	27.7	43	FCK-3-C-N1	FCK-L-3-C FCK-M-3-C FCK-H-3-C	47	27.7	24	M20×1.5	68
	FCK-5-N1	FCK-L-5 FCK-M-5 FCK-H-5	20	37	62	FCK-5-C-N1	FCK-L-5-C FCK-M-5-C FCK-H-5-C	32	37	32	M25×1.5	99
	FCK-8.1-N1	FCK-L-8.1 FCK-M-8.1 FCK-H-8.1	35	37	86	FCK-6.5-C-N1	FCK-L-6.5-C FCK-M-6.5-C FCK-H-6.5-C	50	37	32	M25×1.5	154
	FCK-20-N1	FCK-M-20 FCK-H-20	38	41.6	123	FCK-8.1-C-N1	FCK-L-8.1-C FCK-M-8.1-C FCK-H-8.1-C	55	37	32	M27×1.5	135
	FCK-40-N1	FCK-M-40 FCK-H-40	45	53.1	286	FCK-20-C-N1	FCK-M-20-C FCK-H-20-C	58	41.6	36	M30×1.5	188
						FCK-40-C-N1	FCK-M-40-C FCK-H-40-C	65	53.1	46	M36×1.5	413

1 When using a stopper nut, please note the following points.

● For types without cap, attach the stopper nut so that it protrudes 0.1mm to 1 mm outward along the Shock Absorbers rod from the piston body (cylinder top). For capped types, attach the stopper nut so that it protrudes 0.5 mm to 1 mm plus the cap length outward along the Piston Rod from the shock absorber body (cylinder section).

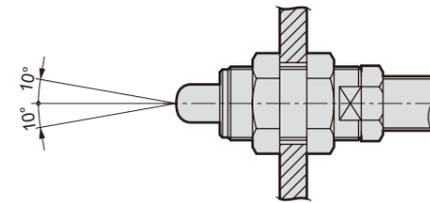


● Fix with the standard nut for Shock Absorbers after installing the stopper nut.

● Cannot be used with a deflection angle adaptor.

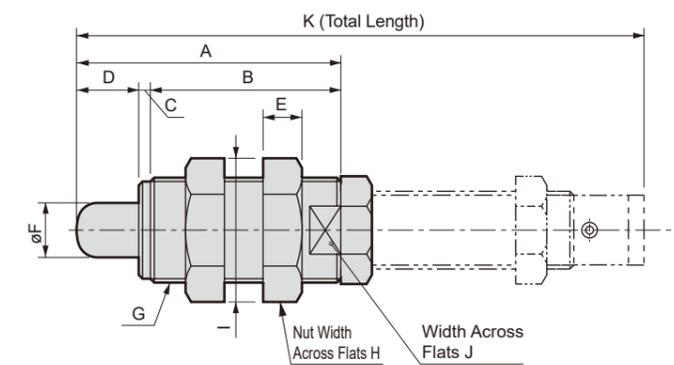
## Specifications

Max Operating Offset Angle  $\pm 10^\circ$



## Dimensional Drawings

● FCK-□-A



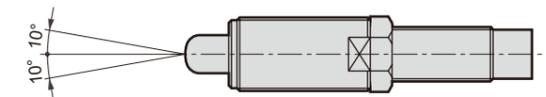
Model No.	Applicable Models	A	B	C	D	E	F	G	H	I	J	K	Tip Part Material	Weight (g)
FCK-0.18-A	FCK-L-0.15 FCK-M-0.18 FCK-H-0.18	38	28	2	8	-	8	M16×1.5	19	20	13	75.7	Polyacetal	37
FCK-0.5-A	FCK-L-0.3 FCK-M-0.5 FCK-H-0.5	48	35	3	10	5	10	M18×1.5	21	24.3	14	97.8		49
FCK-0.6-A	FCK-L-0.4 FCK-M-0.6 FCK-H-0.6	51	38	3	10	7	11	M22×1.5	24	27.7	19	103		83
FCK-1-A	FCK-L-1 FCK-M-1 FCK-H-1	60	45	3	12	7	12	M22×1.5	24	27.7	19	129	81	
FCK-3-A	FCK-L-3 FCK-M-3 FCK-H-3	68	49	3	16	10	14	M27×1.5	32	37	24	146	Ferrous	214
FCK-5-A	FCK-L-5 FCK-M-5 FCK-H-5	107.5	67.5	10	30	15	16	M36×1.5	46	53.1	32	212		630
FCK-8.1-A	FCK-L-8.1 FCK-M-8.1 FCK-H-8.1	97	62	10	25	15	16	M36×1.5	46	53.1	32	188		582
FCK-20-A	FCK-M-20 FCK-H-20	127	82	10	35	15	18	M40×1.5	50	57.7	36	255		838
FCK-40-A	FCK-M-40 FCK-H-40	167	107	10	50	15	20	M45×1.5	55	63.5	41	322	1265	

1 When using an offset angle adapter, please note the following points.

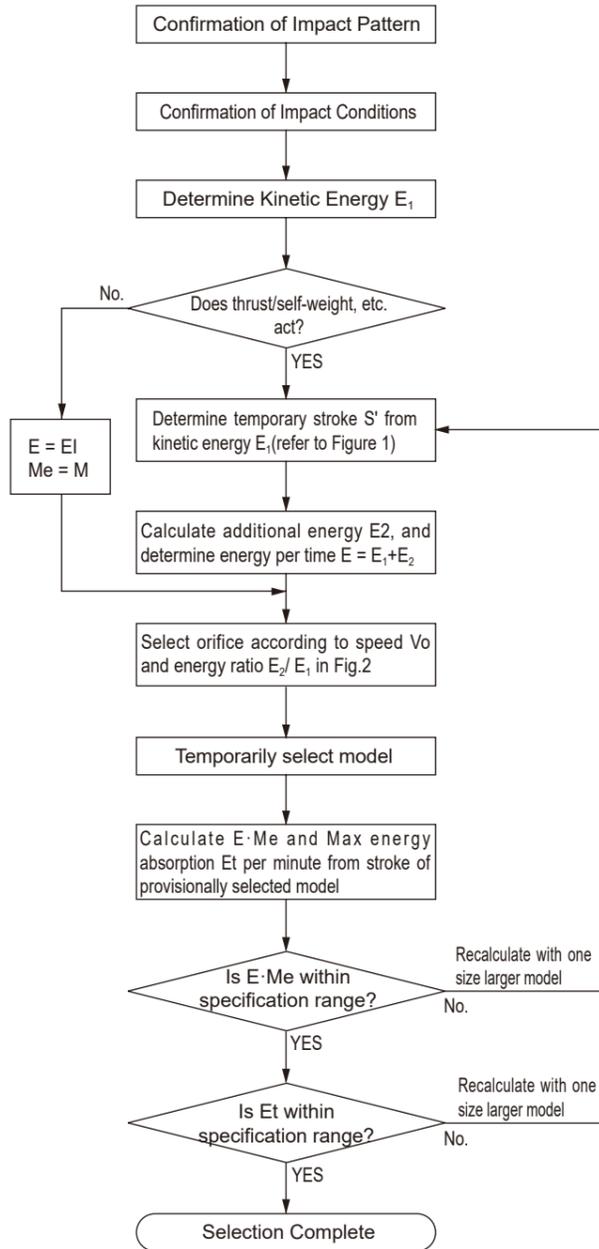
● Keep the angle within  $\pm 10^\circ$  of the center line of the deflection angle adaptor cap.

● Cannot be used with stopper nut.

● Cannot be used with cap.



Model Selection Flowchart



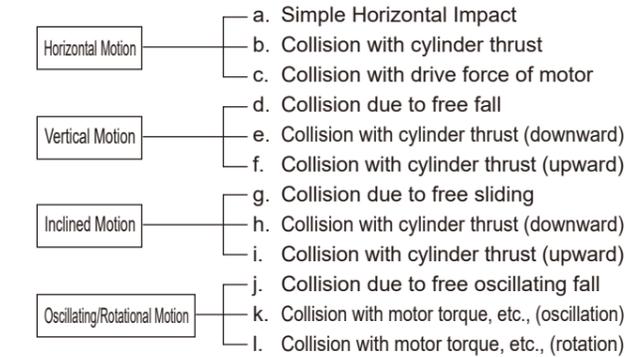
Code	Operating Conditions	Unit
E	Absorption Energy	J
E <sub>1</sub>	Kinetic Energy	J
E <sub>2</sub>	Thrust/Self-weight Energy	J
G	Center of Gravity Position	
S	FCK Stroke	m
g	Gravitational Acceleration (9.8)	m/s <sup>2</sup>
N	Rotations	rpm
Me	Equivalent Weight	kg
Td	Motor Starting Torque	N·m
K	Reduction Ratio	

Impact Pattern Diagram Examples

Usage Example	Horizontal Impact		
	a. Simple Horizontal Impact	b. When there is cylinder thrust	c. When there is motor driving force
Kinetic Energy E <sub>1</sub> (J)	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{1}{2} \cdot M \cdot V^2$
Thrust/Self-generated Energy E <sub>2</sub> (J)	—	$E_2 = F \cdot S$	$E_2 = 2 \cdot \frac{K}{D} \cdot Td \cdot S$
Total Absorption Energy E (J)	$E = E_1$	$E = E_1 + E_2$	$E = E_1 + E_2$
Equivalent Weight Me (kg)	$Me = M$	$Me = \frac{2 \cdot E}{V^2}$	$Me = \frac{2 \cdot E}{V^2}$
Absorption Energy per Hour Et (J/h)	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$
Usage Example	Vertical Impact		
	d. Free Fall	e. Cylinder Lower Limit Stopper	f. Cylinder Upper Limit Stopper
Kinetic Energy E <sub>1</sub> (J)	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{1}{2} \cdot M \cdot V^2$
Thrust/Self-generated Energy E <sub>2</sub> (J)	$E_2 = M \cdot g \cdot S$	$E_2 = (M \cdot g + F) \cdot S$	$E_2 = (F - M \cdot g) \cdot S$
Total Absorption Energy E (J)	$E = E_1 + E_2$	$E = E_1 + E_2$	$E = E_1 + E_2$
Equivalent Weight Me (kg)	$Me = \frac{2 \cdot E}{V^2} (V = \sqrt{2 \cdot g \cdot H})$	$Me = \frac{2 \cdot E}{V^2}$	$Me = \frac{2 \cdot E}{V^2}$
Absorption Energy per Hour Et (J/h)	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$
Usage Example	Inclined Impact		
	g. Free Fall	h. When there is cylinder thrust	i. When there is cylinder thrust
Kinetic Energy E <sub>1</sub> (J)	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{1}{2} \cdot M \cdot V^2$
Thrust/Self-generated Energy E <sub>2</sub> (J)	$E_2 = M \cdot g \cdot S \cdot \sin\theta$	$E_2 = (M \cdot g \cdot \sin\theta + F) \cdot S$	$E_2 = (F - M \cdot g \cdot \sin\theta) \cdot S$
Total Absorption Energy E (J)	$E = E_1 + E_2$	$E = E_1 + E_2$	$E = E_1 + E_2$
Equivalent Weight Me (kg)	$Me = \frac{2 \cdot E}{V^2} (V = \sqrt{2 \cdot g \cdot L \cdot \sin\theta})$	$Me = \frac{2 \cdot E}{V^2}$	$Me = \frac{2 \cdot E}{V^2}$
Absorption Energy per Hour Et (J/h)	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$
Usage Example	Oscillating Impact		Rotational Impact
	j. Free Fall	k. When there is torque from motor, etc.	l. When there is torque from motor, etc.
Kinetic Energy E <sub>1</sub> (J)	$E_1 = M \cdot g \cdot H$	$E_1 = \frac{J \cdot \omega^2}{2}$ or $\frac{1}{2} \cdot M \cdot V^2$	$E_1 = \frac{J \cdot \omega^2}{2} = \frac{M \cdot D^2 \cdot \omega^2}{16}$
Thrust/Self-generated Energy E <sub>2</sub> (J)	$E_2 = \frac{r}{R} \cdot M \cdot g \cdot S$	$E_2 = \frac{T}{R} \cdot S$	$E_2 = \frac{T}{R} \cdot S$
Total Absorption Energy E (J)	$E = E_1 + E_2$	$E = E_1 + E_2$	$E = E_1 + E_2$
Equivalent Weight Me (kg)	$Me = \frac{2 \cdot E}{V^2} (V = \frac{R}{r} \sqrt{\frac{3 \cdot g \cdot H}{2}})$	$Me = \frac{2 \cdot E}{V^2} (V = \omega \cdot R)$	$Me = \frac{2 \cdot E}{V^2} (V = \omega \cdot R, \omega = \frac{2\pi \cdot N}{60})$
Absorption Energy per Hour Et (J/h)	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$	$E_1 = 60 \cdot E \cdot n$

Shock Absorber Model Selection Guide

1 Clarify the equipment impact



\*1: Refer to "Impact Pattern Diagram Examples."

2 Clarify conditions/items necessary for energy calculation

Horizontal Impact			Vertical Impact		
Code	Operating Conditions	Unit	Code	Operating Conditions	Unit
M	Impact Material Weight	kg	M	Impact Material Weight	kg
V	Collision speed	m/s	V	Collision speed	m/s
F	Pushing Force	N	F	Pushing Force	N
n	Repetition Frequency	cycles/min	n	Repetition Frequency	cycles/min
t	Ambient Temperature	°C	t	Ambient Temperature	°C
RT	Return time	s	RT	Return time	s
Inclined Impact			Vibrational/Rotational Impact		
Code	Operating Conditions	Unit	Code	Operating Conditions	Unit
M	Impact Material Weight	kg	M	Impact Material Weight	kg
V	Collision speed	m/s	V	Collision speed	m/s
F	Pushing Force	N	T	Torque	N·m
n	Repetition Frequency	cycles/min	n	Repetition Frequency	cycles/min
t	Ambient Temperature	°C	t	Ambient Temperature	°C
RT	Return time	s	RT	Return time	s
L	Impact Object Travel Distance	m	ω	Angular Velocity	rad/s
θ	Inclination Angle	deg	J	Moment of inertia	kg·m <sup>2</sup>
			R	Distance from Center of Rotation to Impact Point	m
			r	Distance from Center of Rotation to Center of Gravity	m
			α·β	Inclination Angle	deg
			H	Fall Height	m
			D	Diameter of Rotating Body	m

3 Calculation of kinetic energy E<sub>1</sub> according to "Impact Pattern Diagram Examples"

● Calculate kinetic energy E<sub>1</sub> according to "Example of colliding pattern" (P. 1422).

$$E = \frac{1}{2} \cdot M \cdot V^2 = \frac{1}{2} \times 15 \times 1.42^2 = 15.1 \text{ J}$$

4 Select temporary stroke from temporary selection graph

● Fig. 1 (P. 1425) and select the temporary stroke.

S'=30

Selection Examples

Clarify the equipment impact pattern.

Usage Example	Vertical Impact	
	e. Cylinder Lower Limit Stopper	
Kinetic Energy E <sub>1</sub> (J)	$E_1 = \frac{1}{2} \cdot M \cdot V^2$	
Thrust/Gravity Energy E <sub>2</sub> (J)	$E_2 = (Mg + F) \cdot S$	
Total Absorption Energy E (J)	$E = E_1 + E_2$	
Equivalent Weight Me (kg)	$Me = \frac{2 \cdot E}{V^2}$	
Absorption Energy per Hour Et (J/h)	$E_1 = 60 \cdot E \cdot n$	

Colliding object weight : M=15 kg  
 Colliding speed : V=1.42 m/s  
 Pushing force : F=245.5 N  
 Frequency : n=10 cycles/Min  
 Ambient Temperature : t=23°C  
 Return time : Rt=2s (time up to re-collision)

5 Calculation of absorption energy E according to "Impact Pattern Diagram Examples"

- Calculate thrust/self-weight energy  $E_2$  according to "Example of colliding pattern". S (FCK stroke) in the calculation formula is calculated with the temporary stroke S' selected in item 4. Calculate absorption energy E according to "Impact Pattern Diagram Examples."

Selection Examples

$$E_2 = (M \cdot g + F) \cdot S = (15 \times 9.8 + 245.5) \times 0.03 = 11.8 \text{ J}$$

$$E = E_1 + E_2 = 15.1 + 11.8 = 26.9 \text{ J}$$

6 Shock Absorber Temporary Selection

- Select orifice type according to energy ratio (thrust/self-weight energy, kinetic energy) and colliding speed on Fig. 2 (P. 1425), then select a model provisionally according to calculated absorbed energy E.

\*1: Allowable energy absorption may vary depending on colliding speed. Please refer to P. 1426, 1427.

7 Recalculation of absorption energy E with temporarily set model

- Calculate absorbed energy  $E_2$  according to "Example of colliding pattern". S (FCK stroke) during calculation is calculated with the stroke of the model selected in item 6.

$$\frac{E_2}{E_1} = \frac{11.8}{15.1} = 0.8$$

From models with  $E=26.9$  or more  
Temporarily select multi-hole orifice (FCK-H-3)

$$E_2 = (15 \times 9.8 + 245.5) \times 0.016 = 6.28 \text{ J}$$

$$E = 15.1 + 6.28 = 21.4 \text{ J}$$

8 Calculation of Energy per Hour Et

- Calculate energy per hour Et according to "Example of colliding pattern".

$$E_t = 60 \cdot E \cdot n = 60 \times 21.4 \times 10 = 1284 \text{ J/h}$$

9 Confirmation of Equivalent Weight M

- Calculate equivalent weight M according to "Example of colliding pattern".

$$M_e = \frac{2E}{V^2} = \frac{2 \times 21.4}{1.42^2} = 21.2 \text{ kg}$$

10 Selection Confirmation

- If calculated absorbed energy, energy per hour, equivalent weight, frequency of usage, Ambient Temperature and return time are in accordance with specifications of the selected Shock Absorbers, there is no problem. If outside the specification range, select a shock absorber one rank larger than the previously selected model and recalculate.

\*1: The specified equivalent weight depends on the speed. For details, refer to P. 1426, 1427.

[Note]

The impact speed used in shock absorber selection calculation is the speed immediately before impact with the shock absorber. It is different from the cylinder average speed (cylinder stroke / travel time). When performing selection calculation, calculate or actually measure the speed immediately before impact, or use a value 1.5 to 2 times the average speed.

Figure 1. Temporary Selection Graph

Determine temporary stroke S' from kinetic energy  $E_1$ .

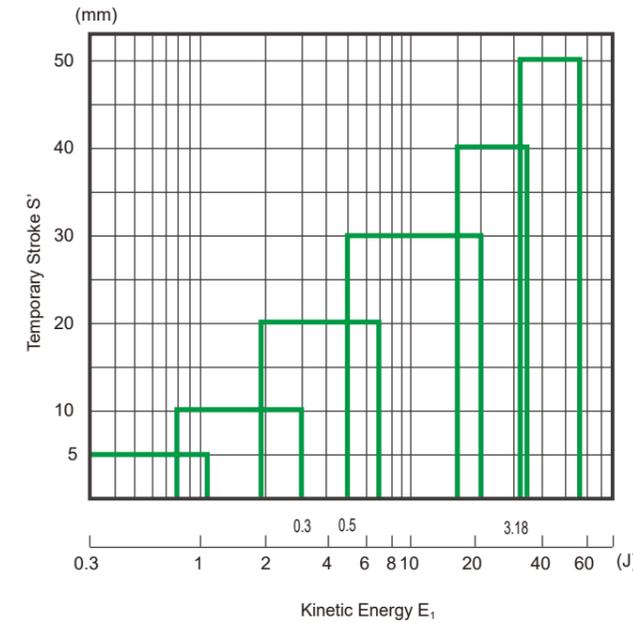
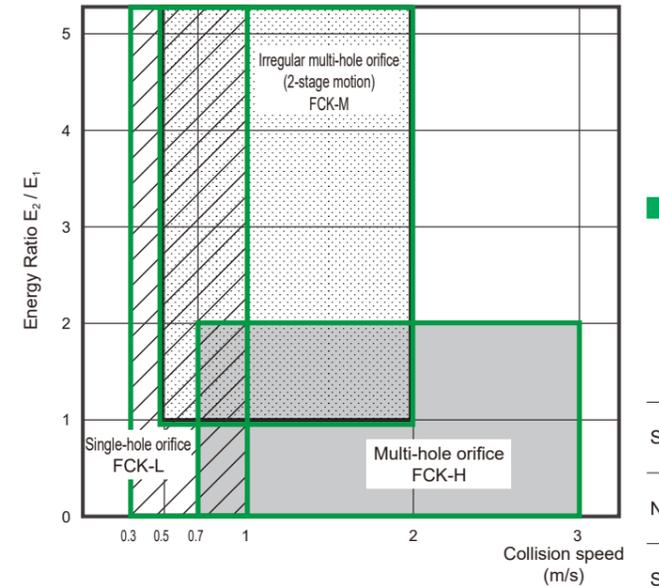


Figure 2. Energy Ratio (Thrust/Self-weight Energy  $E_2$  / Inertial Energy  $E_1$ )

Select orifice type from this figure



Absorption Characteristic Structure

Structure Type	Model	Description	Force-Stroke (F-S) Characteristic
Constant Orifice	FCK-L	Single-hole orifice structures include a dashpot structure utilizing the clearance between the piston and cylinder tube, a single tube structure with an orifice in the piston, and a double tube type single-hole orifice structure (adjustable type), all of which exhibit similar resistance force characteristics. Here, the single tube structure will be explained as a representative. It is structured so that a piston slides inside an oil-filled cylinder tube, and a single-hole orifice is provided in the piston. Since the orifice area is constant over the entire stroke, the absorption characteristic is such that the resistance force immediately after impact is large, and as the stroke progresses and the speed decreases, the resistance force also decreases, as shown in the right figure.	Graph showing a sharp initial peak in force that rapidly decays as stroke increases.
Displacement-dependent Orifice	FCK-H	It has a double structure of an outer tube and an inner tube, and the piston slides on the inner wall of the inner tube. This inner tube is provided with multiple orifices along the stroke direction. As the stroke progresses and the speed decreases, the orifice area decreases stepwise, so the resistance force fluctuates in a ripple pattern, but the Max resistance force can be kept low. Also, it is easy to match the absorption characteristics to individual impact conditions by designing the orifice.	Graph showing a force profile with multiple small peaks (ripples) that remain relatively constant in magnitude throughout the stroke.
	FCK-M	Structurally, it is basically the same as the multi-hole orifice above, but by changing the orifice, energy absorption according to the purpose can be performed instead of constant damping force. For example, the orifice adopted in the FCK-M series is designed to absorb kinetic energy in the first half of the stroke and perform speed control in the latter half. Therefore, it ideally absorbs energy against cylinder thrust.	Graph showing a force profile that is high in the first half of the stroke and then drops significantly in the second half.

Ending

Ending

Equivalent Weight/Impact Speed Characteristic Graph

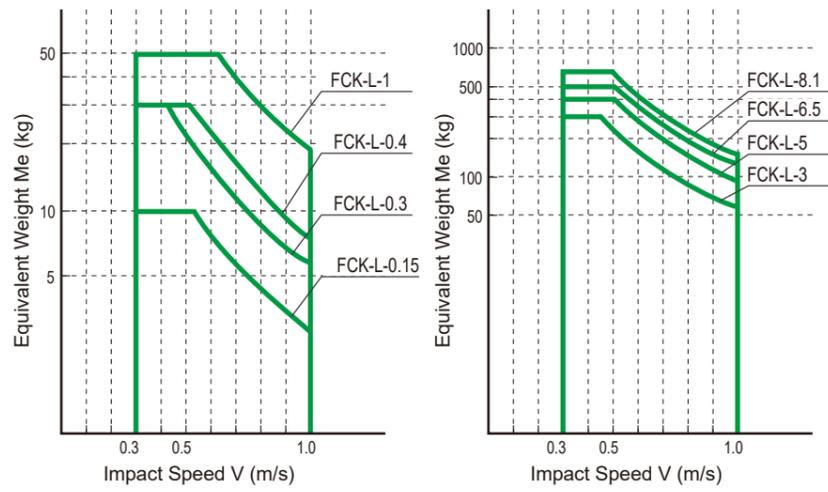
Absorption Energy/Impact Speed Characteristic Graph

Equivalent Weight:  
Weight converted by assuming all cylinder thrust and weight as inertial energy.

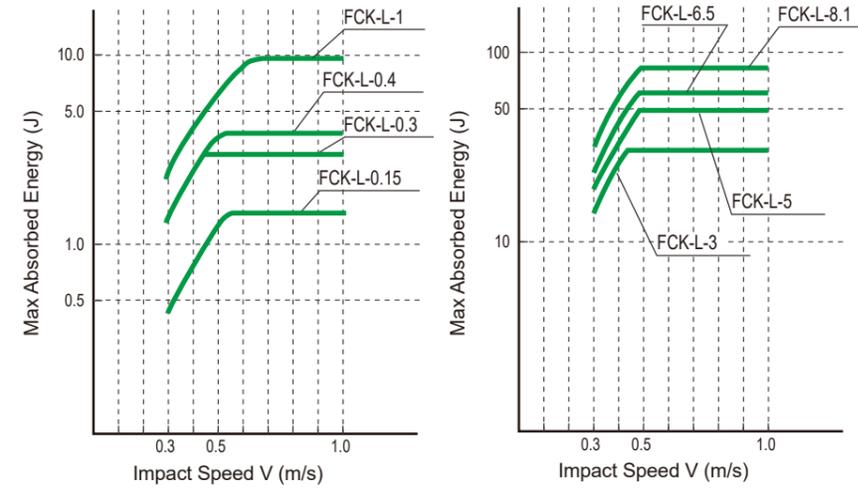
$$\frac{1}{2}MV^2 + F \cdot S = E = MeV^2$$

M: Impact Material Weight  
F: Cylinder thrust or weight self-weight  
Me Equivalent Weight

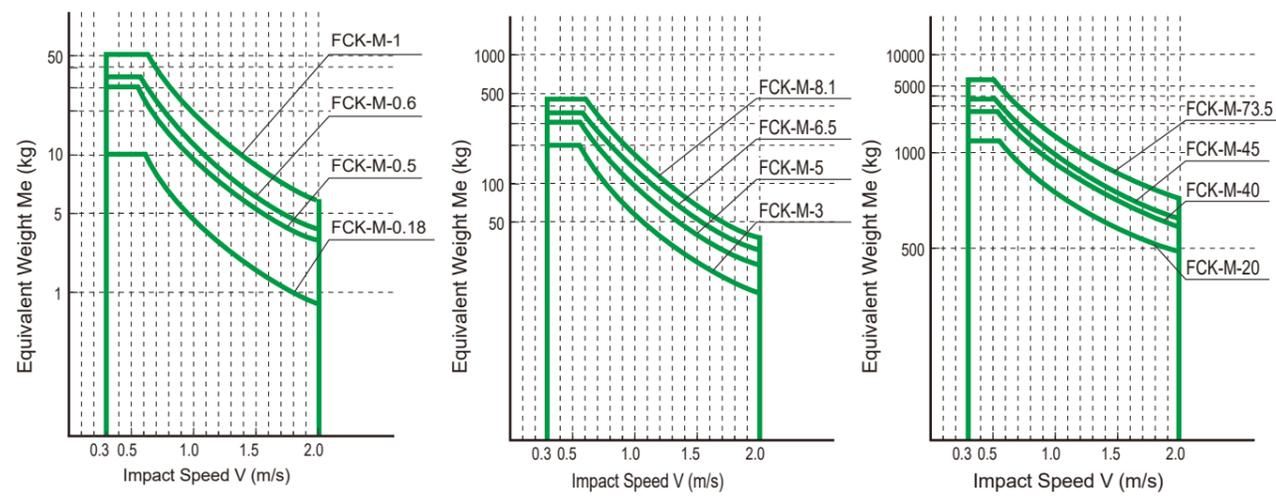
Single-hole Orifice (FCK-L)



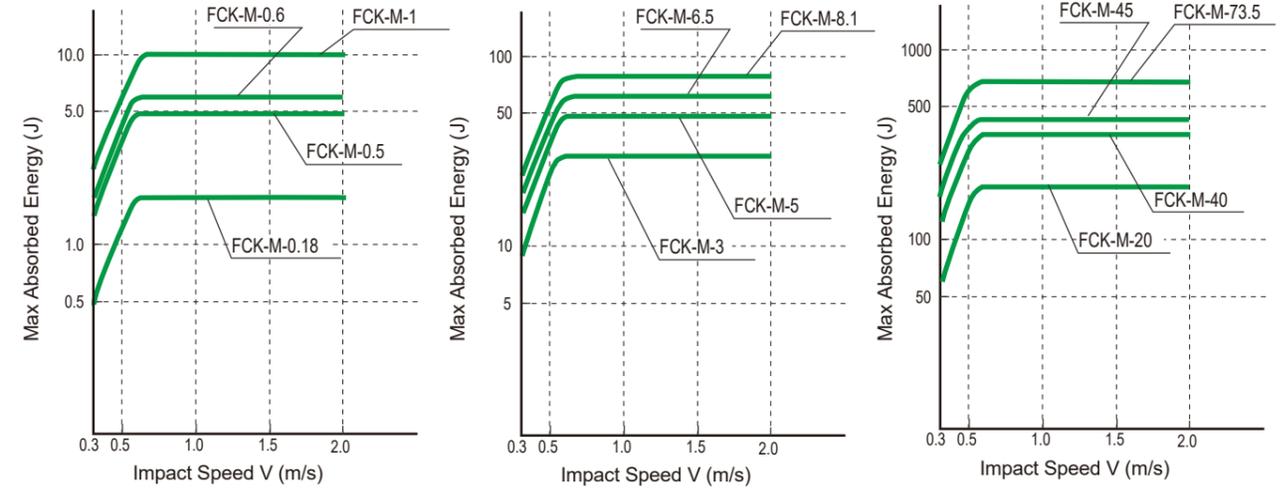
Single-hole Orifice (FCK-L)



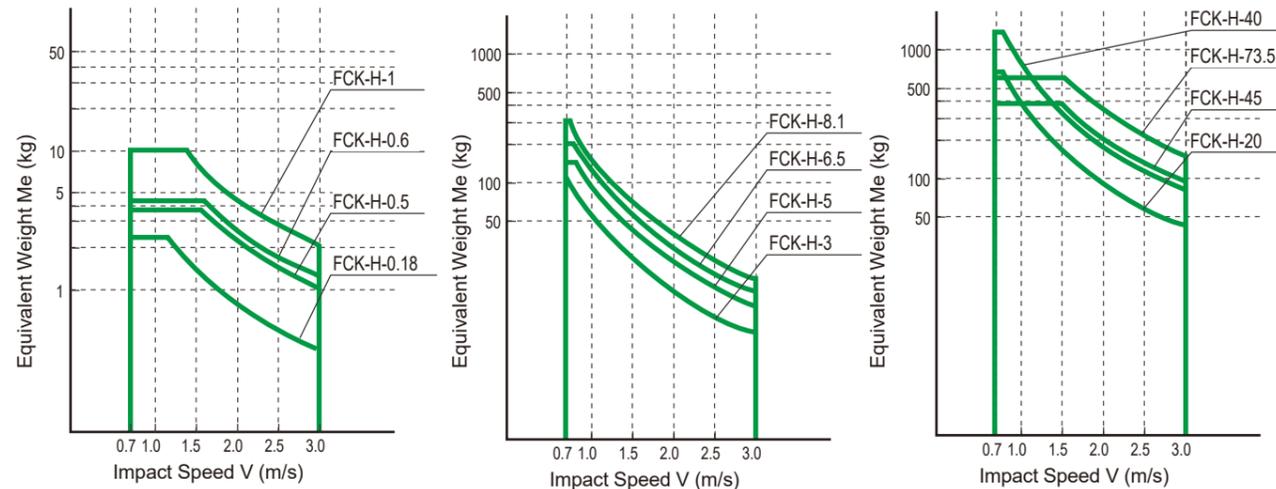
Multi-hole Irregular Orifice (FCK-M)



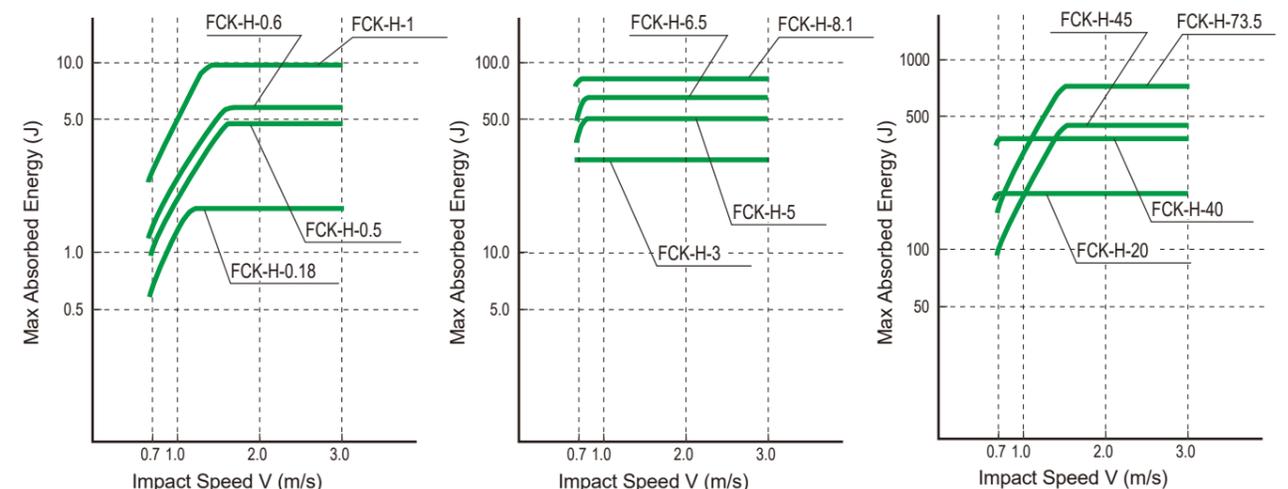
Multi-hole Irregular Orifice (FCK-M)



Multi-hole Orifice (FCK-H)



Multi-hole Orifice (FCK-H)



Ending

Ending

Related Equipment

SKL

NCK

SCK

FCK

FJ

FK

Cylinder Switch

Related Equipment

SKL

NCK

SCK

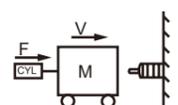
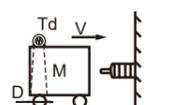
FCK

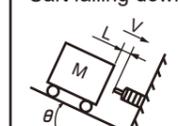
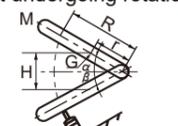
FJ

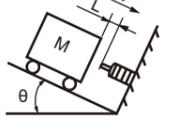
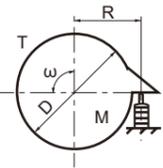
FK

Cylinder Switch

Selection Calculation Example

	Selection Example 1	Selection Example 2
1. Applications	Horizontal impact with cylinder thrust  Cylinder Bore = $\varnothing 40$ Pressure = 0.5 MPa	Horizontal impact with motor driving force  Motor Starting Torque $T_d = 0.196 \text{ N}\cdot\text{m}$ Cart Wheel Diameter $D = 50 \text{ mm}$ Cart Reduction Ratio $K = 10$
2. Colliding conditions	$M = 30 \text{ kg}$ $V = 0.6 \text{ m/s}$ $F = 628.3 \text{ N}$ $(F = \frac{\pi}{4} \times 40^2 \times 0.5 = 628.4 \text{ N})$ $n = 20 \text{ cycles/min}$ $t = 23^\circ\text{C}$ $Rt = 3S$	$M = 150 \text{ kg}$ $V = 0.785 \text{ m/s}$ $F = 78.4 \text{ N}$ $(F = 2 \cdot \frac{K}{D} \cdot T_d = 2 \times \frac{10}{0.05} \times 0.196 = 78.4 \text{ N})$ $n = 5 \text{ cycles/min}$ $t = 23^\circ\text{C}$ $Rt = 2S$
3. Kinetic energy $E_1$	$E_1 = \frac{1}{2} MV^2 = \frac{1}{2} \times 30 \times 0.6^2 = 5.4 \text{ J}$	$E_1 = \frac{1}{2} MV^2 = \frac{1}{2} \times 150 \times 0.785^2 = 46.2 \text{ J}$
4. Temporary stroke $S'$	From Figure 1, $S' = 20 \text{ mm}$	From Figure 1, $S' = 50 \text{ mm}$
5. Thrust/self-weight energy $E_2$ Absorbed energy $E$	$E_2 = F \cdot S = 628.3 \times 0.02 = 12.57 \text{ J}$ $E = E_1 + E_2 = 5.4 + 12.57 = 17.97 \text{ J}$	$E_2 = 2 \cdot \frac{K}{D} \cdot T_d \cdot S = 2 \times \frac{10}{0.05} \times 0.196 \times 0.05 = 3.92 \text{ J}$ $E = E_1 + E_2 = 46.2 + 3.92 = 50.12 \text{ J}$
6. Temporary selection	$\frac{E_2}{E_1} = \frac{12.57}{5.4} = 2.3$ Temporarily select multi-hole irregular orifice (FCK-M-3)	$\frac{E_2}{E_1} = \frac{3.92}{46.2} = 0.08$ Temporarily select multi-hole orifice (FCK-H-6.5)
7. Recalculation of absorbed energy	$E_2 = F \cdot S = 628.3 \times 0.016 = 10.05 \text{ J}$ $E = E_1 + E_2 = 15.45 \text{ J}$	$E_2 = 2 \cdot \frac{K}{D} \cdot T_d \cdot S = 2 \times \frac{10}{0.05} \times 0.196 \times 0.04 = 3.14 \text{ J}$ $E = E_1 + E_2 = 49.34 \text{ J}$
8. Energy per hour $E_t$	$E_t = 60 \cdot E \cdot n = 60 \times 15.45 \times 20 = 18540 \text{ J/h}$	$E_t = 60 \cdot E \cdot n = 60 \times 49.34 \times 5 = 14802 \text{ J/h}$
9. Equivalent weight $M_e$	$M_e = \frac{2E}{V^2} = 85.8 \text{ kg}$	$M_e = \frac{2E}{V^2} = \frac{2 \times 49.34}{0.785^2} = 160 \text{ kg}$
10. Confirmation	$E, E_t, M_e, n, t, Rt$ all OK Decided on FCK-M-3	$E, E_t, M_e, n, t, Rt$ all OK Decided on FCK-H-6.5

Selection Example 3	Selection Example 4
Cart falling down a slope  $L = 1 \text{ m}$ $\theta = 2^\circ$	Object undergoing rotational free fall  $\alpha = 15^\circ$ $\beta = 5^\circ$
$M = 100 \text{ kg}$ $V = 0.83 \text{ m/s}$ $(V = \sqrt{2 \cdot g \cdot L \cdot \sin\theta} = \sqrt{2 \times 9.8 \times 1 \times \sin 2^\circ} = 0.83 \text{ m/s})$ $n = 10 \text{ cycles/min}$ $t = 23^\circ\text{C}$ $Rt = 5S$	$M = 2 \text{ kg}$ $R = 0.5 \text{ m}$ $H = 0.1 \text{ m}$ $r = 0.3 \text{ m}$ $(V = \frac{R}{r} \sqrt{\frac{3 \cdot g \cdot H}{2}} = \frac{0.5}{0.3} \sqrt{\frac{3 \times 9.8 \times 0.1}{2}} = 2.02 \text{ m/s})$ $n = 50 \text{ cycles/min}$ $t = 20^\circ\text{C}$ $Rt = 0.6S$
$E_1 = \frac{1}{2} \cdot M \cdot V^2 = \frac{1}{2} \times 100 \times 0.83^2 = 34.4 \text{ J}$	$E_1 = M \cdot g \cdot H = 2 \times 9.8 \times 0.1 = 1.96 \text{ J}$
From Figure 1, $S' = 50 \text{ mm}$	From Figure 1, $S' = 10 \text{ mm}$
$E_2 = M \cdot g \cdot S \cdot \sin\theta = 100 \times 9.8 \times 0.05 \times \sin 2^\circ = 1.71 \text{ J}$ $E = E_1 + E_2 = 34.4 + 1.71 = 36.1 \text{ J}$	$E_2 = \frac{r}{R} \cdot M \cdot g \cdot S \cdot \cos\beta = \frac{0.5}{0.3} \times 2 \times 9.8 \times 0.01 \times \cos 5^\circ = 0.11 \text{ J}$ $E = E_1 + E_2 = 1.96 + 0.11 = 2.07 \text{ J}$
$\frac{E_2}{E_1} = \frac{1.71}{34.4} = 0.05$ Temporarily select multi-hole orifice (FCK-H-5)	$\frac{E_2}{E_1} = \frac{0.11}{1.96} = 0.06$ Temporarily select multi-hole orifice (FCK-H-0.5)
$E_2 = M \cdot g \cdot S \cdot \sin\theta = 100 \times 9.8 \times 0.03 \times \sin 2^\circ = 1.03 \text{ J}$ $E = E_1 + E_2 = 35.4 \text{ J}$	$E_2 = \frac{r}{R} \cdot M \cdot g \cdot S \cdot \cos\beta = 0.11 \text{ J}$ $E = E_1 + E_2 = 1.96 + 0.11 = 2.07 \text{ J}$
$E_t = 60 \cdot E \cdot n = 60 \times 35.4 \times 10 = 21240 \text{ J/h}$	$E_t = 60 \cdot E \cdot n = 60 \times 2.07 \times 50 = 6210 \text{ J/h}$
$M_e = \frac{2E}{V^2} = \frac{2 \times 35.4}{0.83^2} = 102.7 \text{ kg}$	$M_e = \frac{2E}{V^2} = \frac{2 \times 2.07}{2.02^2} = 1.0 \text{ kg}$
$E, E_t, M_e, n, t, Rt$ all OK Decided on FCK-H-5	$E, M_e, n, t, Rt$ are OK. However, since $E_t$ is over, recalculate with one size larger FCK-H-0.6.

Selection Example 5	Selection Example 6
<p>Object rolling down a slope</p>  <p><math>L = 0.45 \text{ m}</math> <math>\theta = 5^\circ</math></p>	<p>Horizontal rotational impact with torque applied</p> 
<p><math>M = 1.0 \text{ kg}</math> <math>V = 0.88 \text{ m/s}</math> (<math>V = \sqrt{2 \cdot g \cdot L \cdot \sin\theta} = \sqrt{2 \times 9.8 \times 0.45 \times \sin 5^\circ} = 0.88 \text{ m/s}</math>) <math>n = 15 \text{ cycles/min}</math> <math>t = 23^\circ\text{C}</math> <math>Rt = 2\text{S}</math></p>	<p><math>J = 204.1 \text{ kgm}^2</math> <math>\omega = 0.6 \text{ rad/s}</math> <math>R = 1.25 \text{ m}</math> <math>n = 10 \text{ cycles/min}</math> <math>T = 68.6 \text{ N}\cdot\text{m}</math> <math>t = 20^\circ\text{C}</math> <math>Rt = 3\text{s}</math></p>
<p><math>E_1 = \frac{1}{2} \cdot M \cdot V^2 = \frac{1}{2} \times 1.0 \times 0.88^2 = 0.387 \text{ J}</math></p>	<p><math>E_1 = \frac{J \cdot \omega^2}{2} = \frac{204.1 \times 0.6^2}{2} = 36.7 \text{ J}</math></p>
<p>From Figure 1, <math>S' = 5 \text{ mm}</math></p>	<p>From Figure 1, <math>S' = 50 \text{ mm}</math></p>
<p><math>E_2 = M \cdot g \cdot S \cdot \sin\theta = 1 \times 9.8 \times 0.005 \times \sin 5^\circ = 0.004 \text{ J}</math> <math>E = E_1 + E_2 = 0.387 + 0.004 = 0.391 \text{ J}</math></p>	<p><math>E_2 = \frac{T}{R} \cdot S = \frac{68.6}{1.25} \times 0.05 = 2.74 \text{ J}</math> <math>E = E_1 + E_2 = 36.7 + 2.74 = 39.44 \text{ J}</math></p>
<p><math>\frac{E_2}{E_1} = \frac{0.004}{0.387} = 0.01</math> Temporarily select single-hole orifice (FCK-L-0.15)</p>	<p><math>\frac{E_2}{E_1} = \frac{2.74}{36.7} = 0.07</math> <math>V = \omega \cdot R = 0.6 \times 1.25 = 0.75 \text{ m/s}</math> Temporarily select multi-hole orifice (FCK-H-5)</p>
<p><math>E_2 = M \cdot g \cdot S \cdot \sin\theta = 1 \times 9.8 \times 0.008 \times \sin 5^\circ = 0.007 \text{ J}</math> <math>E = E_1 + E_2 = 0.394 \text{ J}</math></p>	<p><math>E_2 = \frac{T}{R} \cdot S = \frac{68.6}{1.25} \times 0.03 = 1.65 \text{ J}</math> <math>E = E_1 + E_2 = 38.6 \text{ J}</math></p>
<p><math>E_t = 60 \cdot E \cdot n = 60 \times 0.394 \times 15 = 354.6 \text{ J/h}</math></p>	<p><math>E_t = 60 \cdot E \cdot n = 60 \times 38.6 \times 10 = 23160 \text{ J/h}</math></p>
<p><math>Me = \frac{2E}{V^2} = \frac{2 \times 0.394}{0.88^2} = 1.02 \text{ kg}</math></p>	<p><math>Me = \frac{2E}{V^2} = \frac{2 \times 38.6}{0.75^2} = 137.2 \text{ kg}</math></p>
<p>E, Et, Me, n, t, Rt all OK Decided on FCK-L-0.15</p>	<p>E, Et, Me, n, t, Rt are OK. Decided on FCK-H-5.</p>

MEMO

Related Equipment

- SKL
- NCK
- SCK
- FCK
- FJ
- FK

Cylinder Switch

Ending

Related Equipment

- SKL
- NCK
- SCK
- FCK
- FJ
- FK

Cylinder Switch

Ending